# **Frequency Domain Specifications**

## What is Frequency Response?

The response of a system can be partitioned into both the transient response and the steady state response. We can find the transient response by using Fourier integrals. The steady state response of a system for an input sinusoidal signal is known as the **frequency response**. In this chapter, we will focus only on the steady state response.

If a sinusoidal signal is applied as an input to a Linear Time-Invariant (LTI) system, then it produces the steady state output, which is also a sinusoidal signal. The input and output sinusoidal signals have the same frequency, but different amplitudes and phase angles.

Let the input signal be -

$$r(t) = A\sin(\omega_0 t)$$

The open loop transfer function will be -

$$G(s) = G(j\omega)$$

We can represent  $G(j\omega)$  in terms of magnitude and phase as shown below.

$$G(j\omega) = |G(j\omega)| \angle G(j\omega)$$

Substitute,  $\omega=\omega_0$  in the above equation.

$$G(j\omega_0) = |G(j\omega_0)| \angle G(j\omega_0)$$

The output signal is

$$c(t) = A|G(j\omega_0)|\sin(\omega_0 t + \angle G(j\omega_0))$$

- The **amplitude** of the output sinusoidal signal is obtained by multiplying the amplitude of the input sinusoidal signal and the magnitude of  $G(j\omega)$  at  $\omega=\omega_0$ .
- The **phase** of the output sinusoidal signal is obtained by adding the phase of the input sinusoidal signal and the phase of  $G(j\omega)$  at  $\omega=\omega_0$ .

Where,

- A is the amplitude of the input sinusoidal signal.
- $\omega_0$  is angular frequency of the input sinusoidal signal.

We can write, angular frequency  $\omega_0$  as shown below.

$$\omega_0 = 2\pi f_0$$

Here,  $f_0$  is the frequency of the input sinusoidal signal. Similarly, you can follow the same procedure for closed loop control system.

# **Frequency Domain Specifications**

The frequency domain specifications are **resonant peak**, **resonant frequency and bandwidth**.

Consider the transfer function of the second order closed loop control system as,

$$T(s) = rac{C(s)}{R(s)} = rac{\omega_n^2}{s^2 + 2\delta\omega_n s + \omega_n^2}$$

Substitute,  $s=j\omega$  in the above equation.

$$T(j\omega) = rac{\omega_n^2}{(j\omega)^2 + 2\delta\omega_n(j\omega) + \omega_n^2} \ \Rightarrow T(j\omega) = rac{\omega_n^2}{-\omega^2 + 2j\delta\omega\omega_n + \omega_n^2} = rac{\omega_n^2}{\omega_n^2\left(1 - rac{\omega^2}{\omega_n^2} + rac{2j\delta\omega}{\omega_n}
ight)} \ \Rightarrow T(j\omega) = rac{1}{\left(1 - rac{\omega^2}{\omega_n^2}
ight) + j\left(rac{2\delta\omega}{\omega_n}
ight)}$$

Let,  $\frac{\omega}{\omega_n}=u$  Substitute this value in the above equation.

$$T(j\omega)=rac{1}{(1-u^2)+j(2\delta u)}$$

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Magnitude of  $T(j\omega)$  is -

$$M=|T(j\omega)|=rac{1}{\sqrt{(1-u^2)^2+(2\delta u)^2}}$$

Phase of  $T(j\omega)$  is -

$$ngle T(j\omega) = -tan^{-1}\left(rac{2\delta u}{1-u^2}
ight)$$

# Resonant Frequency

It is the frequency at which the magnitude of the frequency response has peak value for the first time. It is denoted by  $\omega_r$ . At  $\omega=\omega_r$ , the first derivate of the magnitude of  $T(j\omega)$  is zero.

Differentiate M with respect to u.

$$rac{\mathrm{d}M}{\mathrm{d}u} = -rac{1}{2}ig[(1-u^2)^2 + (2\delta u)^2ig]^{rac{-3}{2}}\,ig[2(1-u^2)(-2u) + 2(2\delta u)(2\delta)ig]$$

$$\Rightarrow rac{\mathrm{d} M}{\mathrm{d} u} = -rac{1}{2}igl[(1-u^2)^2 + (2\delta u)^2igr]^{rac{-3}{2}}\,igl[4u(u^2-1+2\delta^2)igr]$$

Substitute,  $u=u_r$  and  $rac{\mathrm{d}M}{\mathrm{d}u}==0$  in the above equation.

$$egin{aligned} 0 &= -rac{1}{2}ig[(1-u_r^2)^2 + (2\delta u_r)^2ig]^{-rac{3}{2}}\,ig[4u_r(u_r^2-1+2\delta^2)ig] \ &\Rightarrow 4u_r(u_r^2-1+2\delta^2) = 0 \ &\Rightarrow u_r^2-1+2\delta^2 = 0 \ &\Rightarrow u_r^2 = 1-2\delta^2 \ &\Rightarrow u_r = \sqrt{1-2\delta^2} \end{aligned}$$

Substitute,  $u_r=rac{\omega_r}{\omega_n}$  in the above equation.

$$egin{aligned} rac{\omega_r}{\omega_n} &= \sqrt{1-2\delta^2} \ \ \Rightarrow \omega_r &= \omega_n \sqrt{1-2\delta^2} \end{aligned}$$

### Resonant Peak

It is the peak (maximum) value of the magnitude of  $T(j\omega)$ . It is denoted by  $M_r$ .

At  $u=u_r$ , the Magnitude of  $T(j\omega)$  is -

$$M_r = rac{1}{\sqrt{(1-u_r^2)^2 + (2\delta u_r)^2}}$$

Substitute,  $u_r=\sqrt{1-2\delta^2}$  and  $1-u_r^2=2\delta^2$  in the above equation.

$$egin{align} M_r &= rac{1}{\sqrt{(2\delta^2)^2 + (2\delta\sqrt{1-2\delta^2})^2}} \ &\Rightarrow M_r = rac{1}{2\delta\sqrt{1-\delta^2}} \ \end{aligned}$$

Resonant peak in frequency response corresponds to the peak overshoot in the time domain transient response for certain values of damping ratio  $\delta$ . So, the resonant peak and peak overshoot are correlated to each other.

#### Bandwidth

It is the range of frequencies over which, the magnitude of  $T(j\omega)$  drops to 70.7% from its zero frequency value.

At  $\omega = 0$ , the value of u will be zero.

Substitute, u=0 in M.

$$M = rac{1}{\sqrt{(1-0^2)^2 + (2\delta(0))^2}} = 1$$

Therefore, the magnitude of  $T(j\omega)$  is one at  $\omega=0$ .

At 3-dB frequency, the magnitude of  $T(j\omega)$  will be 70.7% of magnitude of  $T(j\omega)$  at  $\omega=0$ .

i.e., at 
$$\omega=\omega_B, M=0.707(1)=rac{1}{\sqrt{2}}$$
  $\Rightarrow M=rac{1}{\sqrt{2}}=rac{1}{\sqrt{(1-u_b^2)^2+(2\delta u_b)^2}}$   $\Rightarrow 2=(1-u_b^2)^2+(2\delta)^2u_b^2$ 

Let, 
$$u_b^2=x$$
 
$$\Rightarrow 2=(1-x)^2+(2\delta)^2x$$
 
$$\Rightarrow x^2+(4\delta^2-2)x-1=0$$
 
$$\Rightarrow x=\frac{-(4\delta^2-2)\pm\sqrt{(4\delta^2-2)^2+4}}{2}$$

Consider only the positive value of x.

$$x=1-2\delta^2+\sqrt{(2\delta^2-1)^2+1}$$
  $\Rightarrow x=1-2\delta^2+\sqrt{(2-4\delta^2+4\delta^4)}$  Substitute,  $x=u_b^2=rac{\omega_b^2}{\omega_n^2}$   $rac{\omega_b^2}{\omega_n^2}=1-2\delta^2+\sqrt{(2-4\delta^2+4\delta^4)}$   $\Rightarrow \omega_b=\omega_n\sqrt{1-2\delta^2+\sqrt{(2-4\delta^2+4\delta^4)}}$ 

Bandwidth  $\omega_b$  in the frequency response is inversely proportional to the rise time  $t_r$  in the time domain transient response.