

1.6. MOHR'S CIRCLE

Mohr's circle is a graphical method of finding normal, tangential and resultant stresses on an oblique plane. Mohr's circle will be drawn for the following cases:

- (i) A body subjected to two mutually perpendicular principal tensile stresses of unequal intensities
- (ii) A body subjected to two mutually perpendicular principal stresses which are unequal and unlike (i.e., one is tensile and other is compressive).
- (iii) A body subjected to two mutually perpendicular principal tensile stresses accompanied by a simple shear stress.

1.26.1 Mohr's Circle When A Body Is Subjected To Two Mutually Perpendicular Principal Tensile Stresses Of Unequal Intensities.

Consider a rectangular body subjected to two mutually perpendicular principal tensile stresses of unequal intensities. It is required to find the resultant stress on an oblique plane.

Let σ_1 = Major tensile stress

σ_2 = Minor tensile stress and

θ = Angle made by the oblique plane with the axis of minor tensile stress.

Mohr's Circle is drawn as follows:

Take any point A and draw a horizontal line through A. Take $AB = \sigma_1$ and $AC = \sigma_2$ towards right from A to some suitable scale. With BC as diameter describe a circle. Let O is the centre of the circle. Now through O, draw a line OE making an angle 2θ with OB.

From E, draw ED perpendicular on AB. Join AE. Then the normal and tangential stresses on the oblique plane are given by AD and ED respectively. The resultant stress on the oblique plane is given by AE

From Fig.

Length AD = Normal stress on oblique plane

Length ED = Tangential stress on oblique plane.

Length AE = Resultant stress on oblique plane.

$$\text{Radius of Mohr's circle} = \frac{\sigma_1 - \sigma_2}{2}$$

Angle ϕ = obliquity.

Problem 1.25. The tensile stresses at a point across two mutually perpendicular planes are 120N/mm^2 and 60N/mm^2 . Determine the normal, tangential and resultant stresses on a plane inclined at 30° to the axis of minor stress by Mohr's circle method

Given Data

Major principal stress, $\sigma_1 = 120\text{N/mm}^2$ (tensile)

Minor principal stress, $\sigma_2 = 60\text{N/mm}^2$ (tensile)

Angle of oblique plane with the axis of minor principal stress,

$$\theta = 30^\circ$$

To find

The normal, tangential and resultant stresses

Solution

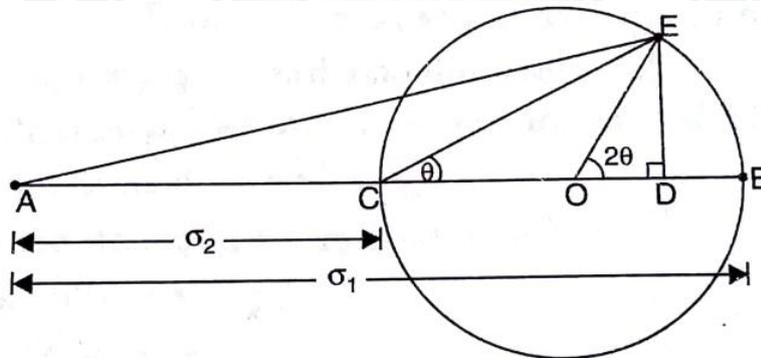
Scale. Let $1\text{cm} = 10\text{N/mm}^2$

Then $\sigma_1 = \frac{120}{10} = 12\text{cm}$ and

$$\sigma_2 = \frac{60}{10} = 6\text{cm}$$

Mohr's circle is drawn as:

Take any point A and draw a horizontal line through A. Take $AB = \sigma_1 = 12\text{cm}$ and AC



$$= \sigma_2 = 6\text{cm}.$$

With BC as diameter (i.e., $BC = 12 - 6 = 6\text{cm}$) describe a circle. Let O is the centre of the circle. Through O, draw a line OE making an angle 2θ (i.e., $2 \times 30 = 60^\circ$) with OB. From E, draw ED perpendicular to CB. Join AE. Measure the length AD, ED and AE.

By measurements :

$$\text{Length AD} = 10.50\text{cm}$$

$$\text{Length ED} = 2.60\text{cm}$$

$$\text{Length AE} = 10.82\text{cm}$$

$$\begin{aligned} \text{Then normal stress} &= \text{Length AD} \times \text{Scale} \\ &= 10.50 \times 10 = \mathbf{105\text{N/mm}^2} \end{aligned}$$

$$\begin{aligned} \text{Tangential or shear stress} &= \text{Length ED} \times \text{Scale} \\ &= 2.60 \times 10 = \mathbf{26\text{ N/mm}^2}. \end{aligned}$$

$$\text{Resultant stress} = \text{Length AE} \times \text{Scale}.$$

$$=10.82 \times 10 = 108.2 \text{ N/mm}^2.$$

1.26.2. Mohr's Circle when a Body is subjected to two Mutually perpendicular Principal stresses which are Unequal and Unlike (i.e., one is Tensile and other is Compressive).

Consider a rectangular body subjected to two mutually perpendicular principal stresses which are unequal and one of them is tensile and the other is compressive. It is required to find the resultant stress on an oblique plane.

Let σ_1 = Major principal tensile stress

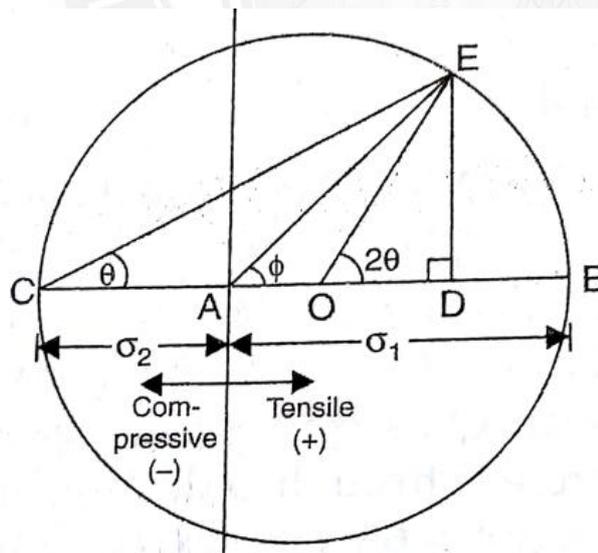
σ_2 = Minor principal compressive stress and

θ = Angle made by the oblique plane with the axis of minor tensile stress.

Mohr's Circle is drawn as follows:

Take any point A and draw a horizontal line through A on both sides of A as shown in fig. Take $AB = \sigma_1 (+)$ towards right of A and $AC = \sigma_2 (-)$ towards left of A to some suitable scale. Bisect BC at O. With O as centre and radius equal to CO or OB, draw a circle. Through O draw a line OE making an angle 2θ with OB.

From E, draw ED perpendicular to AB. Join AE and CE. Then the normal and tangential stresses on the oblique plane are given by AD and ED respectively. The resultant stress on the oblique plane is given by AE



From Fig.

Length AD = Normal stress on oblique plane

Length ED = Tangential stress on oblique plane.

Length AE = Resultant stress on oblique plane.

$$\text{Radius of Mohr's circle} = \frac{\sigma_1 + \sigma_2}{2}$$

Angle ϕ = obliquity.

Problem 1.26. The stresses at a point in a bar are 200N/mm^2 (tensile) and 100N/mm^2 (compressive). Determine the resultant stress in magnitude and direction on a plane inclined at 60° to the axis of major stress. Also determine the maximum intensity of shear stress in the material at the point.

Given Data

Major principal stress, $\sigma_1 = 200\text{N/mm}^2$

Minor principal stress, $\sigma_2 = -100\text{N/mm}^2$

(-ve sign is due to compressive stress)

Angle of oblique plane with the axis of minor principal stress,

$$\theta = 90^\circ - 60^\circ = 30^\circ$$

To find

The Magnitude and direction Resultant stress and maximum intensity of shear stress

Solution

Scale. Let $1\text{cm} = 20\text{N/mm}^2$

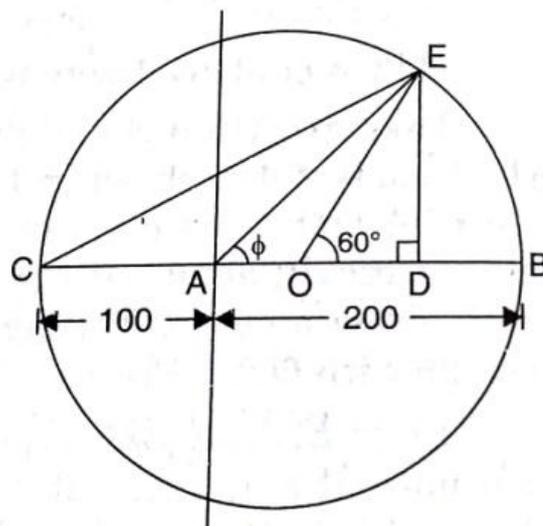
Then $\sigma_1 = \frac{200}{20} = 10\text{cm}$ and

$$\sigma_2 = -\frac{100}{20} = -5\text{cm}$$

Mohr's circle is drawn as:

Take any point A and draw a horizontal line through A on both sides of A as shown in fig. Take $AB = \sigma_1 = 10\text{cm}$ towards right of A and $AC = \sigma_2 = -5\text{cm}$ towards left of A to some suitable scale. Bisect BC at O. With O as centre and radius equal to CO or OB, draw a circle. Through O draw a line OE making an angle 2θ (i.e., $2 \times 30 = 60^\circ$) with OB.

From E, draw ED perpendicular to AB. Join AE and CE. Then the normal and tangential stresses on the oblique plane are given by AD and ED respectively. The resultant stress on the oblique plane is given by AE



By measurements:

$$\text{Length AD} = 6.25\text{cm}$$

$$\text{Length ED} = 6.5\text{cm and}$$

$$\text{Length AE} = 9.0\text{cm}$$

$$\begin{aligned}\text{Then normal stress} &= \text{Length AD} \times \text{Scale} \\ &= 6.25 \times 20 = \mathbf{125\text{N/mm}^2}\end{aligned}$$

$$\begin{aligned}\text{Tangential or shear stress} &= \text{Length ED} \times \text{Scale} \\ &= 6.5 \times 20 = \mathbf{130\text{ N/mm}^2}.\end{aligned}$$

$$\begin{aligned}\text{Resultant stress} &= \text{Length AE} \times \text{Scale.} \\ &= 9 \times 20 = \mathbf{180\text{N/mm}^2}\end{aligned}$$

1.26.3. Mohr's Circle when a Body is subjected two mutually perpendicular principal Tensile Stresses Accompanied by a simple shear stress.

Consider a rectangular body subjected to two mutually perpendicular principal tensile stresses of unequal intensities accompanied by a simple shear stress. It is required to find the resultant stress on an oblique plane.

Let σ_1 = Major tensile stress

σ_2 = Minor tensile stress and

τ = Shear stress across face BC and AD

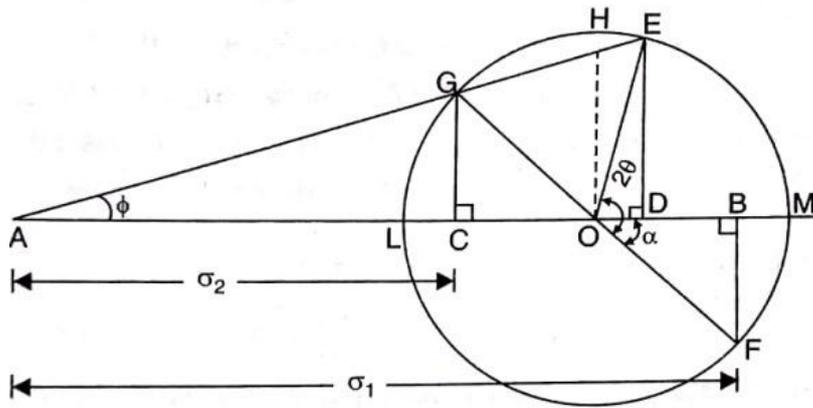
θ = Angle made by the oblique plane with the axis of minor tensile stress.

According to the principle of shear stress, the faces AB and CD will also be subjected to a shear stress of τ

Mohr's Circle is drawn as follows:

Take any point A and draw a horizontal line through A. Take $AB = \sigma_1$ and $AC = \sigma_2$ towards right from A to some suitable scale. Draw perpendiculars at B and C and cut off BF and CG equal to shear stress τ to the same scale. Bisect BC at O. Now with O as centre and radius equal to OG or OF draw a circle. Through O, draw a line OE making an angle 2θ with OF as shown in Fig.

From E, draw ED perpendicular on CB. Join AE. Then the normal and tangential stresses on the oblique plane are given by AD and ED respectively. The resultant stress on the oblique plane is given by AE



From Fig.

Length AD = Normal stress on oblique plane

Length ED = Tangential stress on oblique plane.

Length AE = Resultant stress on oblique plane.

$$\text{Radius of Mohr's circle} = \frac{\sigma_1 - \sigma_2}{2}$$

Angle ϕ = obliquity

Problem 1.27. A rectangular block of material is subjected to a tensile stress of 65N/mm^2 on one plane and a tensile stress of 35N/mm^2 on the plane right angles on the former. Each of the above stresses is accompanied by a shear stress of 25N/mm^2 . Determine the Normal and Tangential stress a plane inclined at 45° to the axis of major stress.

Given Data

Major principal stress, $\sigma_1 = 65\text{N/mm}^2$

Minor principal stress, $\sigma_2 = 35\text{N/mm}^2$

Shear stress, $\tau = 25\text{N/mm}^2$

Angle of oblique plane with the axis of minor principal stress,

$$\theta = 90^\circ - 45^\circ = 45^\circ$$

To Find

The Normal stress and Tangential stress.

Solution

Scale. Let $1\text{cm} = 10\text{N/mm}^2$

Then $\sigma_1 = \frac{65}{10} = 6.5\text{cm}$,

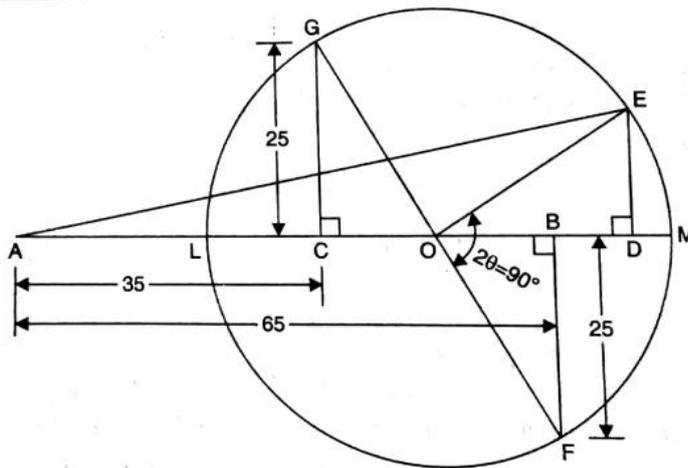
$$\sigma_2 = \frac{35}{10} = 3.5\text{cm and}$$

$$\tau = \frac{25}{10} = 2.5\text{cm}$$

Mohr's circle is drawn as:

Take any point A and draw a horizontal line through A on both sides of A as shown in fig. Take $AB = \sigma_1 = 6.5\text{cm}$ and $AC = \sigma_2 = 3.5\text{cm}$ towards right of A to some suitable scale. Draw perpendiculars at B and C and cut off BF and CG equal to shear stress $\tau = 2.5\text{cm}$ to the same scale. Bisect BC at O. Now with O as centre and radius equal to OG or OF draw a circle. Through O, draw a line OE making an angle 2θ (i.e. $2 \times 45 = 90$) with OF as shown in Fig.

From E, draw ED perpendicular on CB. Join AE. Then the normal and tangential stresses on the oblique plane are given by AD and ED respectively. The resultant stress on the oblique plane is given by AE



By measurements :

Length AD = 7.5 cm and

Length ED = 1.5 cm

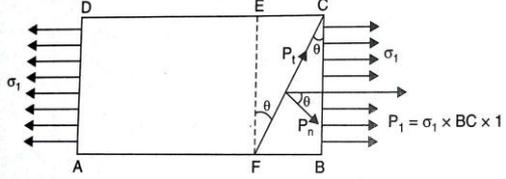
Then normal stress = Length AD \times Scale
 = $7.5 \times 10 = 75 \text{N/mm}^2$

Tangential or shear stress = Length ED \times Scale
 = $1.5 \times 10 = 15 \text{N/mm}^2$.

IMPORTANT TERMS

Stress(σ)	$\sigma = \frac{p}{A}$	$p = \text{Load}$ $A = \text{area of cross section}$
Strain(e)	$e = \frac{dl}{l}$	$dl = \text{change in length}$ $l = \text{original length}$
Lateral strain	$= \frac{dd}{d} = \frac{dt}{t} = \frac{db}{b}$	$d = \text{diameter}$ $t = \text{thickness}$ $b = \text{width}$
Young's Modulus(E)	$E = \frac{\sigma}{e}$	$\sigma = \text{stress}$ $e = \text{strain}$
Shear modulus (or) Modulus of rigidity(C)	$C = \frac{\tau}{\phi}$	$\tau = \text{shear stress}$ $\phi = \text{shear strain}$

Total change in length of a bar	$dl = \frac{p}{E} \left[\frac{L_1}{A_1} + \frac{L_2}{A_2} + \frac{L_3}{A_3} + \dots \right]$	For same material (E = same) with different length and diameter
Total change in length	$dl = p \left[\frac{L_1}{A_1 E_1} + \frac{L_2}{A_2 E_2} + \dots \right]$	For different material with different length and diameter
For composite bar	Total load $P = p_1 + p_2 + \dots$ Strain $e_1 = e_2 = \dots$ Change in length are same	
Total change in length of uniform taper rod	$dl = \frac{4 P L}{\pi E d_1 d_2}$	P = load act on the section L = length of the section E = Young's modulus d_1, d_2 = lager & smaller dia.
Total change in length of uniform taper rectangular bar	$dl = \frac{P L}{E t (a - b)} \log_e \frac{a}{b}$	t = thickness of bar a = width at bigger end b = width at smaller end
Factor of safety	$F.S = \frac{\text{Ultimate Stress}}{\text{Working Stress}}$	
Poisson's ratio(μ)	$\mu = \frac{\text{Lateral strain}}{\text{Linear strain}}$	$e_{latl} = \mu \times e$
For three dimensional stress system	$e_1 = \frac{\sigma_1}{E} - \mu \frac{\sigma_2}{A_2} - \mu \frac{\sigma_3}{A_3}$	Similar for other direction
Total change in length due to self-weight	$dl = \frac{w l^2}{2E}$	w = weight per unit volume of bar
Volumetric strain(e_v)	$e_v = \frac{dl}{l} (1 - 2\mu)$	For one dimension rectangular bar
=	$e_v = \frac{1}{E} (\sigma_x + \sigma_y + \sigma_z) (1 - 2\mu)$	For three dimension cuboid $\sigma_x = \frac{\text{Load in } x \text{ direction}}{\text{Area in } x \text{ direction}}$ Similar for σ_y, σ_z
	$e_v = \frac{dl}{l} - 2 \frac{dd}{d}$	For cylindrical rod
Bulk modulus(K)	$K = \frac{\sigma}{\left(\frac{dV}{V} \right)}$	
Relation between elastic constant E, K, C	$E = 3K(1 - 2\mu)$ $E = 2C(1 + \mu)$	
PRINCIPAL STRESSES AND STRAINS		
<i>A member subjected to a direct stress in one plane</i>		
Direct stress(σ)	$\sigma = \frac{p}{A}$	P = load applied
Normal stress	$\sigma_n = \sigma \cos^2 \theta$	$\sigma_n = \tau \sin 2\theta$
Tangential (or) shear stress	$\sigma_t = \frac{\sigma}{2} \sin 2\theta$	$\sigma_t = -\tau \cos 2\theta$

Max. Normal stress	$= \sigma$		A = area of cross section θ = angle of oblique plane with the normal cross section of the bar
Max. shear (or) Tangential stress	$= \frac{\sigma}{2}$		 <p style="text-align: center;">$\tau = \text{shear stress}$</p>
A member subjected to two like stress in mutually perpendicular direction			
Normal stress	$\sigma_n = \frac{\sigma_1 + \sigma_2}{2} + \frac{\sigma_1 - \sigma_2}{2} \cos 2\theta$		$\sigma_1 = \text{Major tensile stress}$ $\sigma_2 = \text{Minor tensile stress}$ $\theta = \text{angle of oblique plane with the normal cross section of the bar}$
Tangential (or) shear stress	$\sigma_t = \frac{\sigma_1 - \sigma_2}{2} \sin 2\theta$		
Resultant stress	$\sigma_R = \sqrt{\sigma_n^2 + \sigma_t^2}$		<u>When compressive stress put – ve sign</u>
Position of obliquity	$\phi = \tan^{-1} \frac{\sigma_t}{\sigma_n}$		<u>When tensile force is given, we have to find tensile stress = force/that cross section area</u>
Max. shear stress	$(\sigma_t)_{max} = \frac{\sigma_1 - \sigma_2}{2}$		
A member subjected to two like stress in mutually perpendicular direction with shear stress			
Normal stress	$\sigma_n = \frac{\sigma_1 + \sigma_2}{2} + \frac{\sigma_1 - \sigma_2}{2} \cos 2\theta + \tau \sin 2\theta$		$\sigma_1 = \text{Major tensile stress}$ $\sigma_2 = \text{Minor tensile stress}$ $\theta = \text{angle of oblique plane with the normal cross section of the bar}$
Tangential (or) shear stress	$\sigma_t = \frac{\sigma_1 - \sigma_2}{2} \sin 2\theta - \tau \cos 2\theta$		<u>When compressive stress put – ve sign</u>
Resultant stress	$\sigma_R = \sqrt{\sigma_n^2 + \sigma_t^2}$		
Position of principal plane	$\tan 2\theta = \frac{2\tau}{\sigma_1 - \sigma_2}$		<u>When tensile force is given, we have to find tensile stress = force/that cross section area</u>
Max. shear (or) Tangential stress	$(\sigma_t)_{max} = \sqrt{\left(\frac{\sigma_1 - \sigma_2}{2}\right)^2 + \tau^2}$		<u>When inclined stress is given it should be resolved into tensile stress and shear stress</u>
Position of max. shear (or) Tangential stress	$\tan 2\theta = \frac{\sigma_2 - \sigma_1}{2\tau}$		
Major principal stress	$\frac{\sigma_1 + \sigma_2}{2} + \sqrt{\left(\frac{\sigma_1 - \sigma_2}{2}\right)^2 + \tau^2}$		
Minor principal stress	$\frac{\sigma_1 + \sigma_2}{2} - \sqrt{\left(\frac{\sigma_1 - \sigma_2}{2}\right)^2 + \tau^2}$		
Mohr,s Circle			

A body subjected to two mutually perpendicular principal tensile stresses

Step1: select suitable scale

Step2: to draw a horizontal line $AB = \sigma_1$

Step3: to draw $AC = \sigma_2$

Step4: draw a circle with BC as diameter with O as centre

Step4: draw a line OE making an angle 2θ with OB

Step5: from E to draw ED perpendicular to AB

Result:

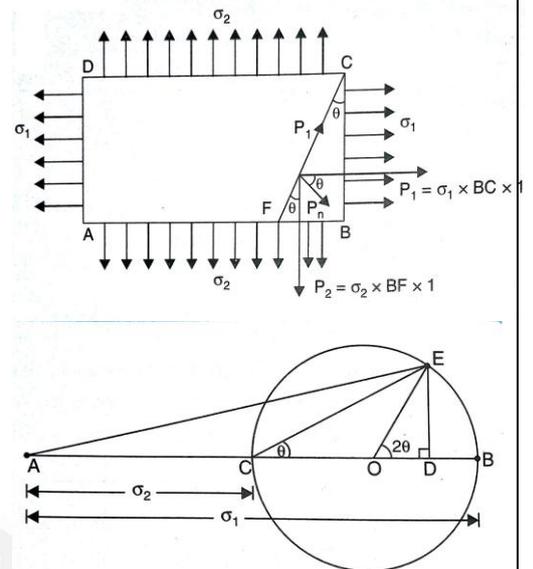
Length AD = Normal stress

Length ED = Tangential (or) shear stress

Length AE = Resultant stress

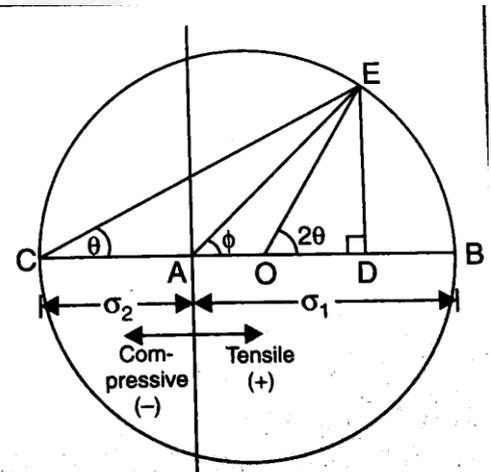
Length OC = OB = Radius of mohr's circle = Max. shear stress

Angle of obliquity = $2\phi = \angle EAD$



A body subjected to two mutually perpendicular principal tensile stresses which are unlike (Tensile and compressive)

All the above procedure are same but step3 will be varied. Because for compressive stress is in -ve sign, hence to draw a line AC in negative direction



A body subjected to two mutually perpendicular principal tensile stresses with simple shear stress

Step1: select suitable scale

Step2: to draw a horizontal line $AB = \sigma_1$

Step3: to draw $AC = \sigma_2$

Step4: draw a perpendicular at B and C as BF and CG = τ

Step5: joint the point G & F which intersect line BC at O.

Step6: draw a circle with O as centre and $OG = OF$ as radius.

Step7: draw a line OE making an angle 2θ with OF

Step8: from E to draw ED perpendicular to AB

Result:

Length AD = Normal stress

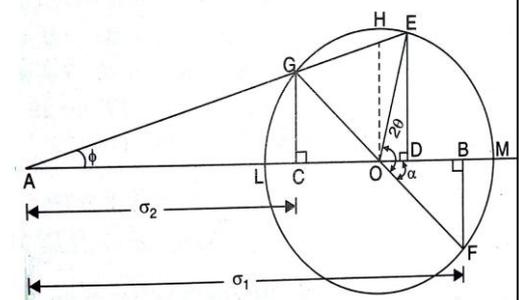
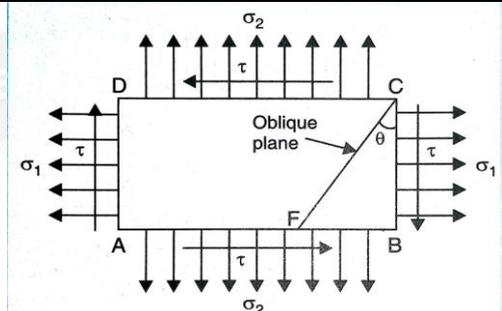
Length ED = Tangential (or) shear stress

Length AE = Resultant stress

Length OG = OF = Radius of mohr's circle = Max. shear stress

Angle of obliquity = $2\phi = \angle EAD$

Length AM = Max. Normal stress



Length $AL = \text{Min. Normal stress}$	
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THEORETICAL QUESTIONS**TWO MARKS:**

1. Define stress and its types
2. Define strain.
3. Define tensile stress and tensile strain.
4. Define the three Elastic moduli.
5. Define shear strain and Volumetric strain

