## STEEL ROOF TRUSSES - ROOFING ELEMENTS

4.2 Design of steel roof truss

Example 2

Design a steel roof truss to suit the following data,

| Span of the truss | $=10 \mathrm{~m}$ |
| :--- | :--- |
| Type of truss | $=$ pan type |
| Roof cover | $=$ Galvanization corrugated (GC) sheeting |
| Materials | $=$ Rolled steel angles |
| Spacing of roof truss | $=4.5 \mathrm{~m}$ |
| Wind pressure | $=1 \mathrm{KN} / \mathrm{m}^{\wedge} 2$ |

Draw the elevation of the roof truss and the details of joints.

Solution:

Step:1 Dimension of truss

$$
\begin{aligned}
\text { Central rise } & =\text { span } / 4 \\
& =10 / 4 \\
& =2.5 \mathrm{~m}
\end{aligned}
$$

Purlins are provided at intervals of 1.863 m on the principal rafter


Fig.4.1 Roof truss

Step:2 Dead loads

Self weight of GC sheeting per purlin at $0.18 \mathrm{KN} / \mathrm{m}^{\wedge} 2$

$$
\begin{aligned}
& =0.18 \times 1.863 \\
& =0.335 \mathrm{KNm}
\end{aligned}
$$

Self weight of purlin at $0.1 \mathrm{KN} / \mathrm{m}$

$$
=0.10 \mathrm{KN} / \mathrm{m}
$$

Total dead load $\quad=0.435 \mathrm{KN} / \mathrm{m}$

Step:3 Live loads
Slope of the truss $=26^{\circ} 34^{-}$
Live load of the truss

$$
\begin{aligned}
& =0.75-(10 \times 0.01+6.5 \times 0.02) \\
& =0.52 \mathrm{KN} / \mathrm{m}^{\wedge} 2
\end{aligned}
$$

Live load per purlin per metre

$$
\begin{aligned}
& =0.52 \times 1.836 x \cos 26 \dot{\circ} 34^{\prime} \\
& =0.87 \mathrm{KN}
\end{aligned}
$$

Step:4 Wind loads

$$
\begin{array}{ll}
\mathrm{F} & =(\mathrm{Cpe}-\mathrm{Cpi}) \mathrm{A} \mathrm{pd} \\
\mathrm{Cpe} & =\text { external pressure coefficient } \\
\mathrm{Cpi} & =\text { internal pressure coefficient } \\
\mathrm{A} & =\text { Surface area of structural element or cladding unit } \\
\mathrm{pd} & =\text { design wind pressure }
\end{array}
$$

Slopping angle ,

$$
\theta=2634
$$

$$
\begin{aligned}
\text { Cpe } & =-0.7 \\
\mathrm{Cpi} & =0.2 \\
\mathrm{~F} & =(-0.7-0.2) \mathrm{pd} \\
& =-0.9 \mathrm{pd} \\
& =-0.9 \mathrm{x} 1 \\
& =-0.9 \mathrm{Kn} / \mathrm{m}^{\wedge} 2
\end{aligned}
$$

Maximum wind load per purlin per metre

$$
\begin{aligned}
& =\left(-0.9 \times 1.863 x \cos 26^{\circ} 34^{\circ}\right) \\
& =1.5 \mathrm{KN}
\end{aligned}
$$

Step:5 Design of purlin

For continuous purlin, the max factored bending moment and shear force are computed as follow,

$$
\begin{aligned}
\mathrm{M} & =\left(1.5 \times 1.305 \times 4.5^{\wedge} 2\right) / 10 \\
& =3.96 \mathrm{KNm} \\
\mathrm{~V} & =(1.5 \times 1.305 \times 4.5) / 2 \\
& =4.4 \mathrm{KN}
\end{aligned}
$$

Adopt ISA 100x75x8mm having section properties given below,

$$
\mathrm{Zx}=\left(4.38 \mathrm{x} 10^{\wedge} 4\right) \mathrm{mm}^{\wedge} 3
$$

D $=100 \mathrm{~mm}$
$\mathrm{b} \quad=75 \mathrm{~mm}$
$\mathrm{t} \quad=8 \mathrm{~mm}$

IS 800:2007 clause 3.7,
(a) Check for section classification is done by computed the rations,

$$
\begin{aligned}
(\mathrm{b} / \mathrm{t}) & =75 / 8 \\
& =9.37<9.4
\end{aligned}
$$

Hence the section considered as plastic .
(b) Check for shear capacity

$$
\begin{aligned}
\mathrm{Av} & =100 \mathrm{X} 8 \\
& =800 \mathrm{~mm}^{\wedge} 2
\end{aligned}
$$

clause 8.4.1,

$$
\begin{aligned}
(\text { Av fy w/V3 } \gamma \mathrm{mo}) & =(800 \times 250) /\left(\mathrm{V} 3 \times 1.10 \times 10^{\wedge} 3\right) \\
& =105 \mathrm{KN}>4.40 \mathrm{KN}
\end{aligned}
$$

The shear capacity of the section is very large compared to the applied shear force.
(c) Check for moment capacity

$$
\begin{aligned}
\mathrm{Md} & =(\beta b \mathrm{Zx} \text { fy }) / \gamma \mathrm{mo} \\
& =\left(1 \times 4.38 \times 10^{\wedge} 4 \times 250\right) /\left(1.1 \times 10^{\wedge} 6\right) \\
\mathrm{Md} & =9.95 \mathrm{KNm}>3.96 \mathrm{KNm}
\end{aligned}
$$

Step:7 Load on truss
(a) Dead load

Slopping length of rafter,

$$
\begin{aligned}
\mathrm{AD} & =\mathrm{V}\left(5^{\wedge} 2+2.5^{\wedge} 2\right) \\
& =5.59 \mathrm{~m}
\end{aligned}
$$

Spacing of trusses $=4.5 \mathrm{~m} \mathrm{c} / \mathrm{c}$
Weight of GC sheeting on half truss ( plan area ) at $0.18 \mathrm{KN} / \mathrm{m}^{\wedge} 2$

$$
\begin{aligned}
& =4.5 \times 5 \times 0.18 \\
& =4.05 \mathrm{KN}
\end{aligned}
$$

$$
\begin{aligned}
& \text { Weight of purlins }(4 \mathrm{nos}) \text { at } 0.10 \mathrm{KN} / \mathrm{m} \\
&=4 \times 0.1 \times 4.5 \\
&=1.8 \mathrm{KN} \\
&=(\operatorname{span} / 300)+0.05 \\
&=(10 / 300)+0.05 \\
& \text { Self weight of roof truss }=0.083 \mathrm{KN} / \mathrm{m}^{\wedge} 2 \\
& \text { Weight of half roof truss }=0.083 \times 5 \times 4.5 \\
&=1.86 \mathrm{KN} \\
& \text { Total load on half truss }=4.05+1.8+1.86 \\
&=7.71 \mathrm{KN}
\end{aligned}
$$

Dead load on intermediate- panel point

$$
\begin{aligned}
& =7.71 / 3 \\
& =2.57 \mathrm{KN}
\end{aligned}
$$

Dead load on end panel point $=2.57 / 2$

$$
=1.285 \mathrm{KN}
$$

(b) Live loads

$$
\begin{aligned}
\text { Live load on half truss } & =0.52 \times 5 \times 4.5 \\
& =11.7 \mathrm{KN}
\end{aligned}
$$

Live load on intermediate panel point

$$
\begin{aligned}
& =11.7 / 3 \\
& =3.9 \mathrm{KN}
\end{aligned}
$$

Live load on end panel point $=3.9 / 2$

$$
=1.95 \mathrm{KN}
$$

(c) Wind loads

Maximum wind load acting perpendicular to the sloping surface

$$
\begin{aligned}
& =0.9 \times 4.5 \times 5.59 \\
& =-22.63 \mathrm{KN}
\end{aligned}
$$

Wind load on intermediate- panel point

$$
\begin{aligned}
& =-(22.68 / 3) \\
& =-7.5 \mathrm{KN}
\end{aligned}
$$

Wind load on end panel point $=-(7.5 / 2)$

$$
=3.75 \mathrm{KN}
$$

Step:8 Design of truss members
(a) Members $\mathrm{AB}, \mathrm{BC}, \mathrm{CD}$

Maximum service load compressive force

$$
=36.17 \mathrm{KN}
$$

Maximum factored compressive force

$$
\begin{aligned}
& =1.5 \times 36.17 \\
& =54.25 \mathrm{KN}
\end{aligned}
$$

Maximum service load tensile force

$$
=22.95 \mathrm{KN}
$$

Maximum factored tensile force

$$
\begin{aligned}
& =1.5 \times 22.95 \\
& =34.42 \mathrm{KN}
\end{aligned}
$$

$$
\text { Length }(\mathrm{L}) \quad=1.863 \mathrm{~m}
$$

Effective length $(\mathrm{KL})=1.304 \mathrm{~m}$

Try two angle ISA 50x50x6mm placed back to back

$$
\text { Area }(\mathrm{A}) \quad=1136 \mathrm{~mm}^{\wedge} 2
$$

Minimum radius of gyration $(\gamma \mathrm{min})=15.1 \mathrm{~mm}$

$$
\begin{array}{ll}
\text { Slenderness ratio } & =(\mathrm{KL} / \gamma \min ) \\
& =1304 / 15.1 \\
& =86.3<180
\end{array}
$$

Stress reduction factor x for column buckling class (c) corresponding to the slenderness

$$
\begin{aligned}
\text { ratio } 86.3 \text { and fy } & =250 \mathrm{~N} / \mathrm{mm}^{\wedge} 2 \\
x & =0.56
\end{aligned}
$$

:Design compressive stress is computed as,

$$
\begin{aligned}
\text { Fcd } & =\mathrm{x} \mathrm{fy} / \gamma \mathrm{mo} \\
& =(0.56 \times 250) / 1.25 \\
& =112 \mathrm{~N} / \mathrm{mm}^{\wedge} 2
\end{aligned}
$$

Design compressive force is given by,

$$
\begin{aligned}
\operatorname{Pd} & =[\mathrm{A} \mathrm{fcd}] \\
& =(1136 \times 112) / 1000 \\
& =127 \mathrm{KN}>54.25 \mathrm{KN}
\end{aligned}
$$

(b) Member DE

Maximum service load tension

$$
=12.83 \mathrm{KN}
$$

Maximum factored load tension

$$
\begin{aligned}
& =1.5 \times 12.83 \\
& =19.24 \mathrm{KN}
\end{aligned}
$$

Maximum service load compression

$$
=9.57 \mathrm{KN}
$$

Maximum factored load compression

$$
\begin{aligned}
& =1.5 \times 9.57 \\
& =14.35 \mathrm{KN} \\
\text { Effective length } & =3 \mathrm{~m}
\end{aligned}
$$

Try a single angle ISA $50 x 50 x 5 \mathrm{~mm}$ connected by 6 mm thick gusset plate the junction with
two bolts of 16 mm at 50 mm .

$$
\begin{aligned}
\text { Gross area }(\mathrm{A}) & =479 \mathrm{~mm}^{\wedge} 2 \\
\gamma \mathrm{~min} & =15.2 \mathrm{~mm}
\end{aligned}
$$

Using 16mm dia bolts,

$$
\begin{aligned}
\text { Anc } & =[50-18] 5 \\
& =160 \mathrm{~mm}^{\wedge} 2 \\
\mathrm{Ago} & =[50-5] 5 \\
& =225 \mathrm{~mm}^{\wedge} 2 \\
\mathrm{Ag} & =479 \mathrm{~mm}^{\wedge} 2
\end{aligned}
$$

(a) Strength governed by rupture of critical section

Tdn $=[0.9$ Anc fy $/ \gamma \mathrm{mi}]+[\beta$ Ago fy $/ \gamma \mathrm{mo}]$
where,
$B=1.4-0.076(\mathrm{w} / \mathrm{t})(\mathrm{fy} / \mathrm{fu})(\mathrm{bs} / \mathrm{Lc})$

$$
\begin{aligned}
& =1.4-0.076(50 / 5)(250 / 410)(50+25 / 50) \\
\beta & =0.70 \\
\mathrm{Tdn} & =[0.9 \times 160 \times 410 / 1.25]+[0.7 \times 225 \times 250 / 1.10] \times 10^{\wedge} 3 \\
& =83.02 \mathrm{KN}=\mathrm{T}_{0}
\end{aligned}
$$

(b) Strength governed by yielding of gross section

$$
\begin{aligned}
\mathrm{Tdg} & =\mathrm{Ag} \text { fy } / \gamma \mathrm{mo} \\
& =\left(470 \times 250 \times 10^{\wedge} 3\right) / 1.10 \\
& =108.8 \mathrm{KN}
\end{aligned}
$$

(c) Strength governed by block shear

$$
\begin{aligned}
\text { Avg } & =5[50+50] \\
& =500 \mathrm{~mm}^{\wedge} 2 \\
\text { Avn } & =5[50+50]-[1.5 \mathrm{x} 18] \\
& =473 \mathrm{~mm}^{\wedge} 2 \\
\text { Atg } & =[5 \times 25] \\
& =125 \mathrm{~mm}^{\wedge} 2 \\
\text { Atn } & =[(5 \times 25)-(0.5 \times 18)] \\
& =116 \mathrm{~mm}^{\wedge} 2
\end{aligned}
$$

The block shear strength is the smaller of the value of Tdb1 and Tdb2 as computed using the equation given below,

$$
\begin{aligned}
\mathrm{Tdb} 1 & =[\mathrm{Avg} \mathrm{fy} / \mathrm{V} 3 \gamma \mathrm{mo}]+[0.9 \mathrm{Atn} \mathrm{fu} / \gamma \mathrm{mi}] \\
& =[(500 \times 250) /(\mathrm{V} 3 \times 1.1)+(0.9 \times 116 \times 410) / 1.25] \\
& =99.92 \mathrm{KN}
\end{aligned}
$$

$$
\begin{aligned}
\mathrm{Tdb} 2 & =[0.9 \mathrm{Avn} \mathrm{fu} / \mathrm{V} 3 \gamma \mathrm{mi}]+[\mathrm{Atg} \mathrm{fy} / \gamma \mathrm{mo}] \\
& =[(0.9 \mathrm{x} 473 \times 410) /(\mathrm{V} 3 \times 1.25)+(125 \times 250) / 1.10] \\
& =109.12 \mathrm{KN}
\end{aligned}
$$

Hence, $\mathrm{Tdb}=109.12 \mathrm{KN}$
The design shear strength is the least of the three value computed under (a)(b)(c), which are $108.8 \mathrm{KN}, 83.02 \mathrm{KN}, 109.12 \mathrm{KN}$.

The design tensile strength of angle $=83.02 \mathrm{KN}>19.24 \mathrm{KN}$
(C) Member BC ,EB

| Service load compressive force | $=6.95 \mathrm{KN}$ |
| :--- | :--- |
| Factored compressive force | $=1.5 \times 6.95$ |
|  | $=10.42 \mathrm{KN}$ |
| Service load tensile force | $=6.38 \mathrm{KN}$ |
| Factored tensile force | $=1.5 \times 6.38$ |
|  | $=9.57 \mathrm{KN}$ |
| Effective length $(\mathrm{kL})$ | $=0.7 \times 1.6$ |
|  | $=1.12 \mathrm{~m}$ |

Use minimum size angle ISA $50 \times 50 \times 5 \mathrm{~mm}$,

$$
\begin{aligned}
\text { Area }(\mathrm{A}) & =479 \mathrm{~mm}^{\wedge} 2 \\
\gamma \mathrm{~min} & =9.7 \mathrm{~mm} \\
\text { Slenderness ratio }(\lambda) & =1120 / 9.7 \\
& =115
\end{aligned}
$$

The stress reduction factor x corresponding to $\mathrm{fy}=250 \mathrm{~N} / \mathrm{mm}^{\wedge} 2$ and $\lambda=115$

$$
\mathrm{x} \quad=0.39
$$

Design compressive stress is computed as,

$$
\begin{aligned}
\mathrm{fcd} & =\mathrm{x} \mathrm{fy} / \gamma \mathrm{mo} \\
& =(0.39 \times 250) / 1.25 \\
& =78 \mathrm{~N} / \mathrm{mm}^{\wedge} 2
\end{aligned}
$$

Design compressive force is given by ,

$$
\begin{aligned}
\mathrm{Pd} & =\mathrm{A} \mathrm{fcd} \\
& =[479 \times 78] / 1000 \\
& =37.36 \mathrm{KN}>10.42 \mathrm{KN}
\end{aligned}
$$

(d) Member EA and EF

| Max service load tension | $=32.21 \mathrm{KN}$ |
| ---: | :--- |
| Factored tension | $=1.5 \times 32.21$ |
|  | $=748.31 \mathrm{KN}$ |

Max service load compression

|  | $=18.84 \mathrm{KN}$ |
| :--- | :--- |
| Factored compression | $=1.5 \times 18.84$ |
|  | $=28.26 \mathrm{KN}$ |
| Length of member | $=3.33 \mathrm{~m}$ |
| Effective length $(\mathrm{kL})$ | $=0.7 \times 3.33$ |
|  | $=2.331 \mathrm{~m}$ |

Try minimum two angle ISA 50x50x6mm connect by guesst plate 6 mm thick with two 16 mm dia bolts spaced at 50 mm

$$
\begin{aligned}
\text { Area }(\mathrm{A}) & =2 \times 598 \\
& =113.6 \mathrm{~mm}^{\wedge} 2
\end{aligned}
$$

$$
\gamma \min =15.1 \mathrm{~mm}
$$

i) Design strength due to yielding of cross section ,

$$
\begin{aligned}
\mathrm{Tdj} & =\text { Ag fy } / \gamma \mathrm{mo} \\
& =[(1136+250) / 1.10] \times 10^{\wedge}-3 \\
& =258 \mathrm{KN}
\end{aligned}
$$

ii) Design strength governed by tearing at net section,

$$
\mathrm{Tdn}=\alpha \mathrm{An} \mathrm{fu} / \gamma \mathrm{mi}
$$

Assume a single line of 16 mm dia bolts of two number spaced 50 mm apart $\mathrm{x}=0.6$

$$
\begin{aligned}
\text { An } & =[(50-18)(6 \times 2)] \\
& =384 \mathrm{~mm}^{\wedge} 2 \\
\text { Tdn } & =[(0.6 \times 384 \times 410) / 1.25] \times 10^{\wedge}-3 \\
& =75.5 \mathrm{KN}>48.31 \mathrm{KN}
\end{aligned}
$$

Hence, the angle section designed for the truss can safely resist the factored loads.

## Example 2

A beam column is to be designed to support a factored axial load of 500 KN ( tension). Factored moment Mx Acting at top and bottom of the column are 30 KNm and 50 KNm respectively. Effective length of column may be Taken as 3.2 m . Assuming $\mathrm{fy}=250 \mathrm{~N} / \mathrm{mm}^{\wedge} 2$, design the beam column section and check the same to conform the Specification of the Indian standard code IS 800:2007.

Solution:

Given data:

| Factored axial load | $=600 \mathrm{KN}$ (tension) |
| :--- | :--- |
| Bending moment at top | $=30 \mathrm{KNm}$ |
| Bending moment at bottom | $=50 \mathrm{KNm}$ |

Yield stress of steel $\quad=250 \mathrm{~N} / \mathrm{mm}^{\wedge} 2$

Step 1 Selection of beam column section

$$
\begin{aligned}
\mathrm{Mdx} & =\mathrm{Z}_{0} \mathrm{fy} / \gamma \mathrm{mo} \\
& =\left(62 \times 10^{\wedge} 4 \times 250\right) /\left(1.1 \times 10^{\wedge} 6\right) \\
& =140.7 \mathrm{KNm} \\
\mathrm{Tdg} & =\mathrm{fy} \mathrm{Ag} / \gamma \mathrm{mo} \\
& =250 \times 6500 / 1.10 \times 1000 \\
& =1477.3 \mathrm{KN}
\end{aligned}
$$

Desgin strength due to rupture of critical section,

$$
\begin{aligned}
\mathrm{Tdn} & =0.9 \mathrm{fy} \mathrm{An} / \gamma \mathrm{mi} \\
& =(0.9 \times 415 \times 6500) /(1.25 \times 1000) \\
& =1942.2 \mathrm{KN}
\end{aligned}
$$

The design strength $\mathrm{Td}=1477.3 \mathrm{KN}$

Step 2 Check for resistance of cross section to combined effects

Using the interaction equation,

$$
\begin{aligned}
& {\left[\begin{array}{rl}
{[\mathrm{N} / \mathrm{Nd}} & +\mathrm{Mx} / \mathrm{Mdx}+\mathrm{My} / \mathrm{Mdy}] \leq 1.0 \\
\mathrm{Nd} & =\mathrm{Ag} \text { fy } / \gamma \mathrm{mo} \\
& =6500 \mathrm{x} 250 / 1.1 \mathrm{x} 1000 \\
& =1477.3 \mathrm{KN}
\end{array}\right.} \\
& \begin{aligned}
\mathrm{Mx} & =50 \mathrm{KNm} \text { and }
\end{aligned} \\
& \mathrm{Mdx}=140.7 \mathrm{KNm} \\
& \therefore[600 / 1477.3+50 / 140.7]=0.756<1
\end{aligned}
$$

Hence safe

Step 3 : Check for lateral torsional buckling resistance

Reduced effective moment is computed as,

$$
\begin{aligned}
\text { Meff } & =[\mathrm{M}-\Psi \mathrm{Tt} \mathrm{Zec} / \mathrm{A}] \leq \mathrm{Md} \\
& =\left[\left(50 \times 10^{\wedge} 6\right)-\left(0.8 \times 600 \times 10^{\wedge} 3 \times 619 \times 10^{\wedge} 3\right) / 6500\right] \\
& =4.3 \times 10^{\wedge} 6 \mathrm{Nmm} \\
& =4.3 \mathrm{KNm}<127.3 \mathrm{KNm}
\end{aligned}
$$

Step 4 Check for overall buckling strength

$$
\begin{aligned}
& {[\mathrm{P} / \mathrm{Pdx}+\mathrm{Meff} / \mathrm{Mdx}] \leq 1.0} \\
& {[600 / 1477.3+4.3 / 127.3]=0.439<1.0}
\end{aligned}
$$

Hence safe

